

Tangential-Rotation and Normal-Normal-Momentum Continuous Mixed Finite Elements for Non-Linear Shell Models

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12th Austrian Numerical Analysis Day, Innsbruck, April 28-29, 2016



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Continuum mechanics

Deformation $\Phi : \Omega \rightarrow \mathbb{R}^3$

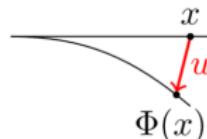
Displacement $u := \Phi - id$

Deformation gradient $F := \nabla \Phi$

Cauchy-Green strain tensor $C := F^T F$

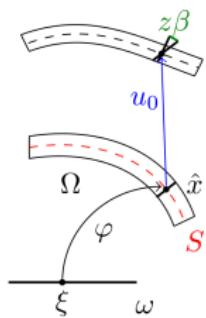
Green strain tensor $E := \frac{1}{2}(C - I)$

Linearized strain tensor $\epsilon(u) := \frac{1}{2}(\nabla u^T + \nabla u)$



$$\frac{\|\Phi(x + \Delta x) - \Phi(x)\|^2}{\|\Delta x\|^2} = \frac{\Delta x^T F^T F \Delta x}{\|\Delta x\|^2} + \mathcal{O}(\|\Delta x\|)$$

Thin-walled structures



- Model of reduced dimensions
- $\Omega = \{\varphi(\xi) + zn(\xi) : \xi \in \omega, z \in [-\frac{t}{2}, \frac{t}{2}]\}$
- $u(\hat{x} + zn(\xi)) = u_0(\hat{x}) + z\beta(\hat{x})$

Geometric derivation

$$\nabla u(\hat{x} + zn) = \nabla_\tau u_0 + z \nabla_\tau \beta + \beta n^T$$

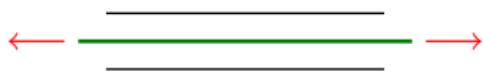
$$C_{\tau\tau} = (P_\tau + \nabla_\tau u_0^T)(P_\tau + \nabla_\tau u_0)$$

$$C_{\tau n} = P_\tau(I + \nabla_\tau u_0)^T(n + \beta)n^T$$

$$C_{\tau\tau} := P_\tau C P_\tau$$

$$C_{\tau n} := P_\tau C P_n$$

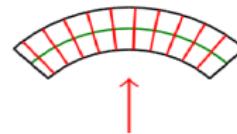
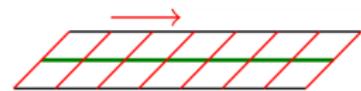
$$C_{nn} := P_n C P_n$$



$$E_{\text{memb}} \approx \|C_{\tau\tau} - I_{\tau\tau}\|_{L_2(S)}^2$$

$$E_{\text{shear}} \approx \|C_{\tau n}\|_{L_2(S)}^2$$

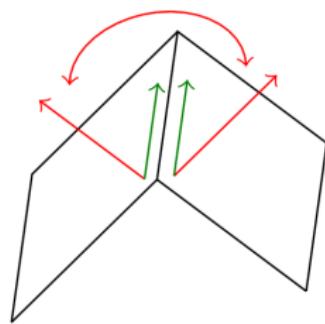
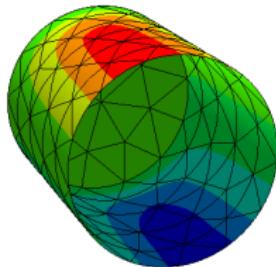
$$E_{\text{bend}} \approx t^2 \| \epsilon_{\tau\tau}((I + \nabla_\tau u_0)^T \beta) \|_{L_2(S)}^2$$



First variational problem

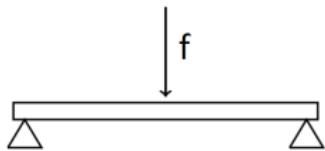
Find u and $\beta \in [H^1(S)]^3$ such that

$$J(u, \beta) = \|C_{\tau\tau} - I_{\tau\tau}\|_{L_2(S)}^2 + \|C_{\tau n}\|_{L_2(S)}^2 + t^2 \|\epsilon_{\tau\tau}((I + \nabla_\tau u_0)^T \beta)\|_{L_2(S)}^2 - \int_S f \cdot u \, dx \rightarrow \min!$$



- β tangential continuous
- u continuous
- m normal-normal continuous

Kirchhoff plate and Hellan-Herrmann-Johnson method



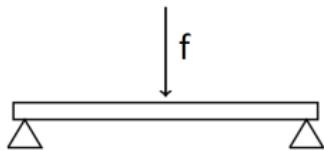
$$\begin{cases} \Delta^2 w = f & \text{in } \Omega \\ w = \frac{\partial w}{\partial n} = 0 & \text{on } \partial\Omega \end{cases}$$

Find $w \in H_0^2(\Omega)$, such that

$$\int_{\Omega} \nabla^2 w : \nabla^2 v \, dx = \int_{\Omega} fv \, dx \quad \text{for all } v \in H_0^2(\Omega).$$

$$\begin{aligned} \int_{\Omega} m : \tau \, dx - \int_{\Omega} \nabla^2 w : \tau \, dx &= 0 & \forall \tau \\ - \int_{\Omega} m : \nabla^2 v \, dx &= - \int_{\Omega} fv \, dx & \forall v \end{aligned}$$

Kirchhoff plate and Hellan-Herrmann-Johnson method



$$\begin{cases} \Delta^2 w = f & \text{in } \Omega \\ w = \frac{\partial w}{\partial n} = 0 & \text{on } \partial\Omega \end{cases}$$

Find $w \in H_0^2(\Omega)$, such that

$$\int_{\Omega} \nabla^2 w : \nabla^2 v \, dx = \int_{\Omega} fv \, dx \quad \text{for all } v \in H_0^2(\Omega).$$

$$\begin{aligned} \int_{\Omega} m : \tau \, dx - \int_{\Omega} \nabla^2 w : \tau \, dx &= 0 & \forall \tau \\ - \int_{\Omega} \text{divdiv} mv \, dx &= - \int_{\Omega} fv \, dx & \forall v \end{aligned}$$

Derivation

$$\int_{\Omega} m : \tau \, dx + \sum_T \int_T \nabla w \cdot \operatorname{div} \tau \, dx - \int_{\partial T} \nabla w \cdot \tau_n \, ds = 0$$
$$\sum_T \int_T \operatorname{div} m \cdot \nabla v \, dx - \int_{\partial T} (\operatorname{div} m) \cdot nv \, ds = - \int_{\Omega} fv \, dx$$

$$\int_E \nabla w [\tau_n]_E \, ds = \int_E \nabla_n w [\tau_{nn}]_E + \nabla_\tau w [\tau_{n\tau}]_E \, ds = \int_E \nabla_\tau w [\tau_{n\tau}]_E \, ds$$
$$\int_{\partial T} (\operatorname{div} m) \cdot nv \, ds = 0 \quad \text{as well as} \quad \int_{\partial T} \nabla_\tau v \cdot m_{n\tau} \, ds = 0$$

Find $m \in H(\operatorname{div} \operatorname{div})$ and $w \in H^1(\Omega)$, such that

$$\int_{\Omega} m : \tau \, dx + \sum_T \int_T \nabla w \cdot \operatorname{div} \tau \, dx - \int_{\partial T} \nabla_\tau w \cdot \tau_{n\tau} \, ds = 0 \quad \forall \tau,$$
$$\sum_T \int_T \operatorname{div} m \cdot \nabla v \, dx - \int_{\partial T} \nabla_\tau v \cdot m_{n\tau} \, ds = - \int_{\Omega} fv \, dx \quad \forall v.$$

[Comodi, 1989]

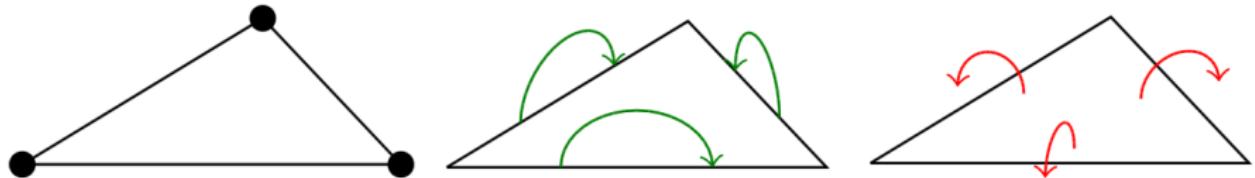
Reissner-Mindlin method

Find $m \in H(\text{divdiv})$, $\beta \in H(\text{curl})$ and $u \in H^1(\Omega)$, such that

$$\int_{\Omega} m : \tau \, dx + \sum_T \int_T \beta \cdot \operatorname{div} \tau \, dx - \int_{\partial T} \beta_{\tau} \cdot \tau_{n\tau} \, ds = 0 \quad \forall \tau,$$
$$\sum_T \int_T \operatorname{div} m \cdot \delta \, dx - \int_{\partial T} \delta_{\tau} \cdot m_{n\tau} \, ds - \frac{1}{t^2} \int_{\Omega} (\nabla u - \beta)(\nabla v - \delta) \, dx = - \int_{\Omega} f \cdot \delta \, dx \quad \forall v, \delta.$$

- Shear strain: $\nabla u - \beta$
- Shear stress: $\frac{1}{t^2}(\nabla u - \beta)$
- For $t \rightarrow 0$: $\nabla u - \beta \rightarrow 0$
- Error analysis similar

Finite Element Spaces



Boundary conditions: $u = u_D$ on Γ_D , $\beta = \beta_D$ on Γ_D , $m_n = m_N$ on Γ_N

| | Displacements | Rotations | Bending moments |
|-----------|---------------------------------|--------------|-----------------|
| Essential | u | β_τ | m_{nn} |
| Natural | $\frac{\partial u}{\partial n}$ | β_n | $m_{n\tau}$ |

$$V_h := \{v \in L_2(\Omega) : v|_T \in \mathcal{P}^{k+1}, v \text{ continuous}, v = 0 \text{ on } \Gamma_D\} \subset H^1$$

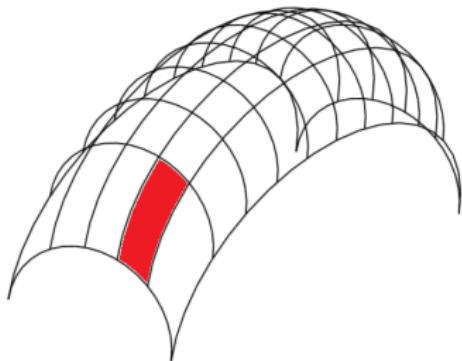
$$B_h := \{\delta \in [L_2(\Omega)]^n : \delta|_T \in \mathcal{P}^k, \delta_\tau \text{ continuous}, \delta_\tau = 0 \text{ on } \Gamma_D\} \subset H(\text{curl})$$

$$M_h := \{\tau \in L_2^{\text{sym}}(\Omega) : \tau|_T \in \mathcal{P}^k, \tau_{nn} \in \mathcal{P}^{\max(1, k-1)} \text{ continuous}, \tau_{nn} = 0 \text{ on } \Gamma_N\} \subset H(\text{divdiv})$$

Finite Element Spaces

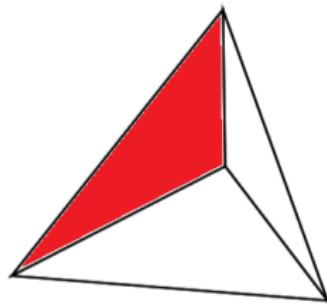
Finite Element Space M_{Surf} :

- 2D elements of M_h as face-elements



Finite Element Space V_h and B_h :

- Traces of 3D elements



Error estimates

Discrete norms

$$\begin{aligned} \|\beta\|_{B_h}^2 &:= \sum_T \|\epsilon(\beta)\|_{L^2(T)}^2 + \sum_{F \in \mathcal{F}} \frac{1}{h_F} \|[\beta_n]\|_{L^2(F)}^2 \\ \|m\|_{M_h}^2 &:= \sum_T \|m\|_{L^2(T)}^2 + \sum_{F \in \mathcal{F}} h_F \|m_{nn}\|_{L^2(F)}^2 \end{aligned}$$

Elasticity

$$\|m - m_h\|_{M_h}^2 + \|\beta - \beta_h\|_{B_h}^2 \leq Ch^{2k} \|\beta\|_{H^{k+1}(\Omega)}^2$$

Reissner-Mindlin

$$\|m - m_h\|_{M_h}^2 + \|\beta - \beta_h\|_{B_h}^2 + \frac{1}{t^2} \|(\nabla u - \beta) - (\nabla u_h - \beta_h)\|_{L^2(\Omega)}^2 \leq C(m, \beta, u) h^{2k}$$

[Schöberl + Pechstein, 2011]

Minimization problem

Find $m \in V$ and $u \in Q$, such that

$$\begin{aligned} a(m, \tau) + b(\tau, u) &= 0 && \text{for all } \tau \in V, \\ b(m, v) &= f(v) && \text{for all } v \in Q. \end{aligned}$$

Lagrange-function

$$L(m, u) = \frac{1}{2}a(m, m) + b(m, u) - f(u)$$

- $L(m, u)$ convex in $m \Rightarrow J(u) := \min_m L(m, u)$ by solving linear problem
- $J(u)$ concave
- Find $\max_u J(u)$ with Newton's method

Shell model

Bilinear forms

$$a(m, \tau) := -\frac{1}{t^2} \int_S m : \tau \, dx$$

$$b(m, v) := \sum_{T \in S} \int_T \operatorname{div} m \cdot ((\nabla v)^T n) \, dx - \int_{\partial T} m_{n\tau} \cdot ((\nabla_\tau v)^T n) \, ds$$

- $a(\cdot, \cdot)$ and $b(\cdot, \cdot)$ correspond to E_{bend}
- Adding $E_{\text{memb}} = \|C_{\tau\tau} - I_{\tau\tau}\|_{L_2(S)}^2$

Shell model

First $E_{\text{shear}} = \frac{1}{t^2} \|(\nabla u)^T n - \beta\|_{L_2(\Omega)}^2 = 0$

Find $u \in [V_h]^3$ and $m \in M_{\text{Surf}}$ such that

$$L(m, u) = -\frac{1}{2}a(m, m) + b(m, u) + E_{\text{memb}}(u) - f(u),$$
$$J(u) = \min_m L(u, m) \rightarrow \min!.$$

Adding $E_{\text{shear}} = \frac{1}{t^2} \|(\nabla u)^T n - \beta\|_{L_2(\Omega)}^2$

Find $u \in [V_h]^3$, $m \in M_{\text{Surf}}$ and $\beta \in B_h$, such that

$$L(m, u, \beta) = -\frac{1}{2}a(m, m) + b(m, \beta) + E_{\text{memb}}(u) + E_{\text{shear}}(u, \beta) - f(u),$$
$$J(u, \beta) = \min_m L(u, \beta, m) \rightarrow \min!.$$

NGS-PY

```
Etautau=Ptau(1/2*(C-I))
```

```
a=BilinearForm(fes,symmetric=True)
a+=SymbolicBFI(-100*InnerProduct(m,tau),BND)
a+=SymbolicBFI(-div(m)*delta-div(tau)*beta,BND)
a+=SymbolicBFI(mn*tang(delta)+taun*tang(beta),element_boundary=True)

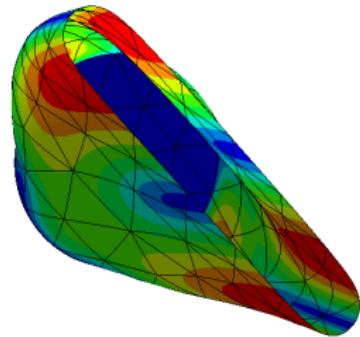
a+=SymbolicEnergy(InnerProduct(Etautau,Etautau),BND)

a+=SymbolicEnergy(100*(ngradu-beta)*(ngradu-beta),BND)

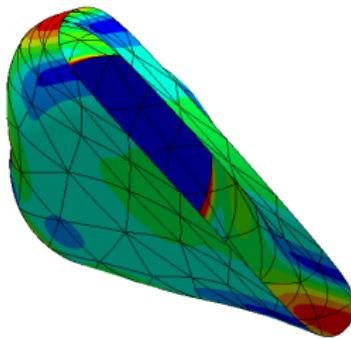
a+=SymbolicEnergy(-y*uy,BND)
```

Apply Newton's method with automatic differentiation

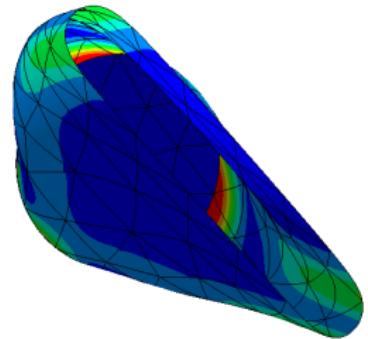
Examples



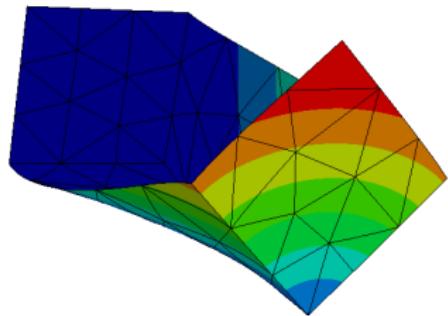
Rotation



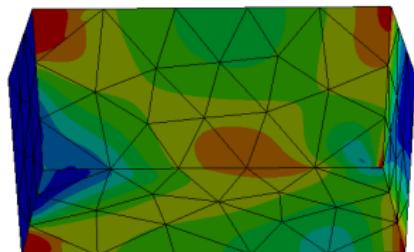
Moments



Shear stress



Displacement



Moments

Thank you for your attention!